# High-performance, Biodegradable Synthetic Ester Lubricants for Railway Systems

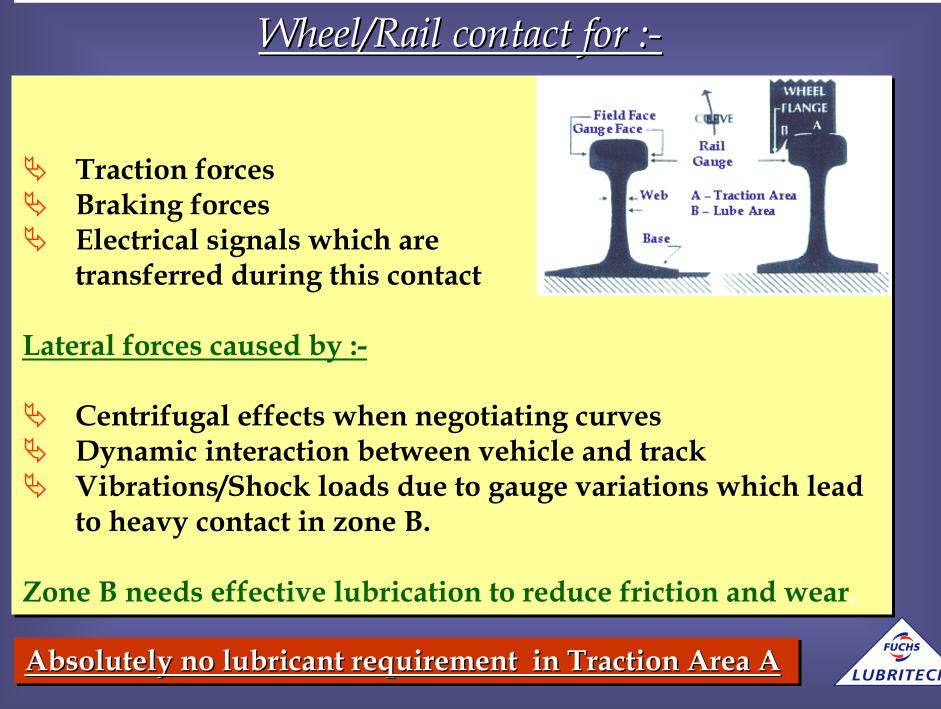




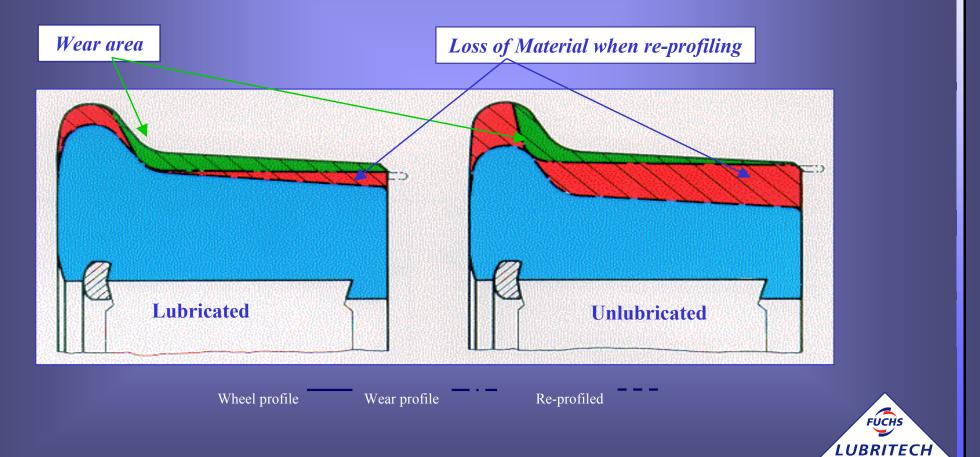
# THE PROBLEM

- Wear of wheel flanges
- Wear of gauge rail face
- Significant maintenance costs attributed to re-profiling
- Loss of availability of the train due to outages for re-profiling wheels
- Screeching on curves due to lack of, or poor lubrication
- Greater track resistance due to un-lubricated contact with the wheel flange radius





# Influence of Lubrication on Wheel Flange Wear



# Wheel Flange Lubrication Requirements

**Primary Requirements:-**

- Reduction of friction between wheel and rail flange to reduce the risk of derailment and to minimise rolling noises generated in the contact zones.
- Reduction of wear of wheel and rail to minimise maintenance work and life cycle costs, and to increase the availability of the rolling stock.

**Secondary Requirements :-**

- Excellent adhesive properties are required and become more important with the introduction of high speed trains.
- Lubricity and sprayability should be independent of temperature change which is not the case with bituminous products.
- **Good resistance against ageing.**



# What lubricant should be selected ?

# The traditional choice was usually based on bitumen, but this can be a health hazard

Bitumen Solvents

# **ALL CAN BE CARCINOGENIC !**

Heavy Metals



# **Bitumen Based Greases**

The apparent viscosity of the grease is totally dependent on temperature and temperature change This behaviour seriously affects the film thickness and subsequent strength; therefore, large volumes of lubricant will be required with poor results.

- @ 50 to 80°C it drips off
- @ 15°C the material hardens
- @ 5 to 8°C depending on quality, it solidifies



# The Ecological Aspect

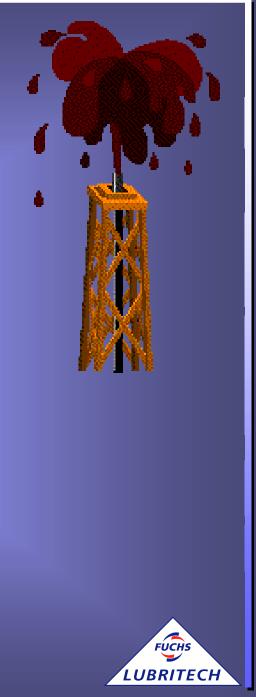
Estimated Oil Resource: 135 billion tonnes





Lubricants in the Environment in Tonnes Per Annum

- Western Europe circa 2,700, 000
- World circa 37,000,000:
- Of which Base Oils are 95%
- And Chemical Additives 5%



# BIODEGRADABLE SYNTHETIC ESTERS?



FUCHS

LUBRITEC

The Solution "High Performance" Biodegradable Synthetic Lubricants

# **Basic formulation of the lubricant :-**

For these applications soft free-flowing greases should be used, which may contain special solid lubricants.

Even though the base oil is by far the biggest portion of the lubricant, it is important that all single components of the additive and thickening system are checked regarding their environmental potential if they are to meet the appropriate standard for biodegradability.

Base oil selection:-

- Synthetic Esters
- Low-viscous Polyglycols
- Low-viscous Synthetic Hydrocarbons

# **Esters - what are they ?**

Very high performance, very long life lubricants developed from a natural resource and alcohols



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# *Objectives For A Wheel Flange Lubricant*

- Environmentally harmless
- Solution No toxic components
- **Low water pollution**
- Solvent-free
- Solution High performance
- 🏷 Long Life
- **High biodegradability**



**Biodegradability test methods are not homogeneously regulated, CEC L-33-A-93 is the standard most commonly used.** 

# The solution is a kind of jigsaw puzzle

Besides biodegradability, the following properties have to be considered when selecting the base oils:-

- 🍤 Lubricity
- S Flow behaviour at low temperatures
- 🍤 Thermal resistance
- 🍤 Solubility
- Solution Miscibility of additives
- Sompatibility with materials to which applied

The following thickening systems can be used in biodegradable lubricants:-

- 🖖 🛛 Lithium soap
- 🍤 Calcium soap
- A blend of Li-/Ca- soap
- 🍤 Inorganic thickeners, e.g. Bentonite



# Additives and Solid Lubricants:-

**EP additives play an important role because wheel flange lubrication is working under boundary or extreme mixed friction conditions.** 

Selection of additives must take into consideration and fulfil all the tribological demands as well as the important environmental aspects.

Long-term experience in this area has shown that solid lubricants can be extremely helpful in these applications. Solid lubricants such as special graphite types, and a combination of white solids and graphite are frequently used.



LUBRITECH

# Solution:- Locolube ECO

| Colour                                 | Dark Grey             |                  |
|--|-----------------------|------------------|
| Temperature Range                      | -30°c to +80°c        | DIN 51 825       |
| Base Oil Viscosity @ 40 <sup>0</sup> c | 35 mm <sup>2/</sup> s | DIN 51 562-1     |
| Pour Point of Base Oil                 | < - 40 <sup>0</sup> c | DIN ISO 3016     |
| Stirring Properties @ - 30°c           | Good Quality<br>Rate  | DB-TL 950.103    |
| Un-worked Penetration (-30°c,24h)      | 240 1/10mm            | DIN ISO 2137     |
| NLGI Grade                             | 000                   | DIN 51 818       |
| Water Resistance                       | 1-60 Value Step       | DIN 51 807       |
| Oil Separation (40°c, 18 H )           | < 2%                  | DB Specification |
| 4-ball Test, Weld Load                 | 2,600/2,800 N         | DIN 51 350-4     |
| Tannert Coefficient of Friction        | 0.12                  | Fuchs Lab. Proc. |
| Biodegradability                       | 90% after 21 Days     | CEC L-33 A 94    |

# Comparison Between Conventional & <u>Locolube ECO</u>

| <u>Data</u>                      | Standard Lube           | Locolube ECO              |
|----------------------------------|-------------------------|---------------------------|
| Base Oil                         | Mineral Oil             | Saturated Synthetic Ester |
| Base Oil Viscosity @ 40°c        | 21.3 mm <sup>2</sup> /s | 35 mm <sup>2/</sup> s     |
| Additives                        | AC,AW,EP                | AC,AW,AO,EP               |
| Solids                           | Graphite                | Graphite & White Solids   |
| Temperature Range                | -25 to + 60°c           | -30 to + 80°c             |
| Thickener                        | Calcium Soap            | Bentonite                 |
| NLGI Grade                       | NLGI 1                  | NLGI 000                  |
| 4 Ball Test Weld Load            | 1,500 N                 | 2,600/2,800 N             |
| Amount Applied Per Cycle         | 30 mg                   | 30 mg                     |
| Cycles Per 500 Metre (546 Yards) | 4                       | 1                         |

AC = Anti-corrosion, AW = Anti-wear, AO = Anti-oxydant, EP = Extreme Pressure



# To enable us to hit the target we have utilized the following tests and testing procedures

| <b>Friction value</b> |          | Environment  |
|-----------------------|----------|--------------|
| Tannert               | ₩        | CEC L-33-A93 |
| Timken                | <b>₩</b> | OECD 301 A-E |
| Almen-Wieland         | <b>₩</b> | RAL-UZ 64    |
| Adhesion power        |          | <u>Wear</u>  |
| Tannert               | ₩\$      | FBT welding  |
| Timken-Retention      |          | load         |
| Fling-off test        | ₩\$      | FBT wear     |
|                       | ₩\$      | Timken       |
| <u>Corrosion p</u>    | orote    | <u>ction</u> |
| 🔖 Erichsen            |          |              |
| 🄖 Water res           | sistan   | nce          |
| Structure Emcor       |          |              |



# Summary of Biodegradable Lubricants for Wheel Flanges

- Biodegradable lubricants have surpassed, technically and environmentally, mineral oils and greases.
   Biodegradable lubricants, due to their performance, have completely replaced mineral oils and greases in many applications due to :-
- A reduction in driving power of up to 15%
   A reduction of wear, noise and friction
   An increase in the maintenance intervals
   A reduction of maintenance activity
   A reduction of lubricant consumption
   A reduction of the risk of derailment



# Lubrication Is a Two-fold Problem

- Choice of lubricant
- Means of application

A wrong choice will lead inevitably

to under-performance and higher costs







# **Wheel Flange Lubrication vs Track Lubrication**

# Wheel Flange Lubrication

- Continuous lubrication of the whole network with very small amounts of lubricant (0.03g/per cycle).
- 100% of lubricant is used for lubrication - no contamination of the track area.
- Self-regulating lubrication management by curve, distance or time, depending on the control unit employed, which is independent of the driver or maintenance team.

# **Track Side Lubrication**

- Re-filling intervals difficult to determine.
- High concentration of lubricant in the curve sections causes vehicle and environmental pollution.
- Correct lubrication of the whole network is not possible.



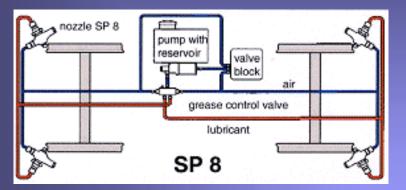
For decades Willy Vogel AG have been producing and supplying different types of wheel-flange lubrication systems for installation on rail vehicles. Vogel wheel-flange systems are renowned for the high degree of metering accuracy, regardless of fluctuations in air pressure and temperature, and this is one of the main advantages of, and reason for, their worldwide acceptance.

Due to the large number of possible combinations and a perfected control technology, it is possible to choose the exact wheel-flange lubrication system for a variety of operating conditions.

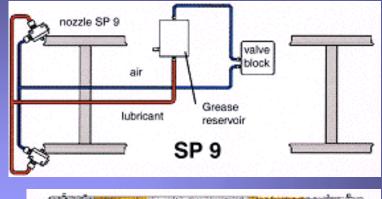
Wheel-flange lubrication, which includes the correct choice of lubricant, is a major factor when it comes to increasing the service life of wheel sets and tracks and cutting life cycle costs. Wheel sets can often cover 8-900,000 km before they have to be re-profiled, so the installation of wheel-flange lubrication systems quite often pays off within a year of installation, without taking into account the savings resulting from longer rail life, particularly in the curved areas.



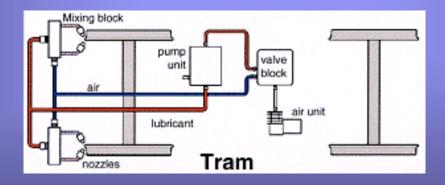
# **Vogel Systems Overview**















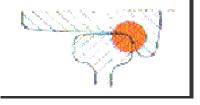


# Application of Wheel Flange Lubricant Utilizing Automatic Lubricators

- Automatic lubricators spray extremely small quantities of lubricant onto the wheel flanges
- The control of the system is divided into:-Path Controlled

# Time Controlled and

**Curved Controlled Systems** 





Lubrication Principle

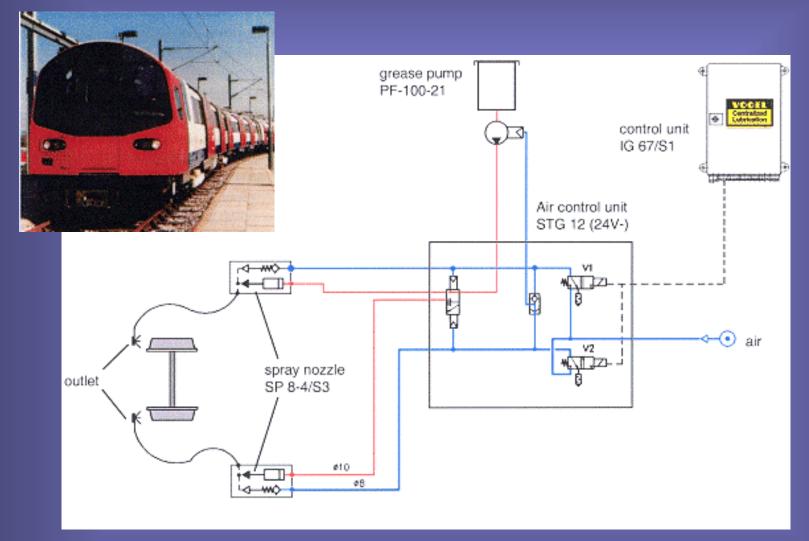
Indirect Lubrication

- Nozzle directs lubricant to the radius of the wheel (not the track!)
- The wheel transfers lubricant to the track
- The track then transfers lubricant to the following wheels



Direct Lubrication

# **Vogel WFL Systems SP 8**

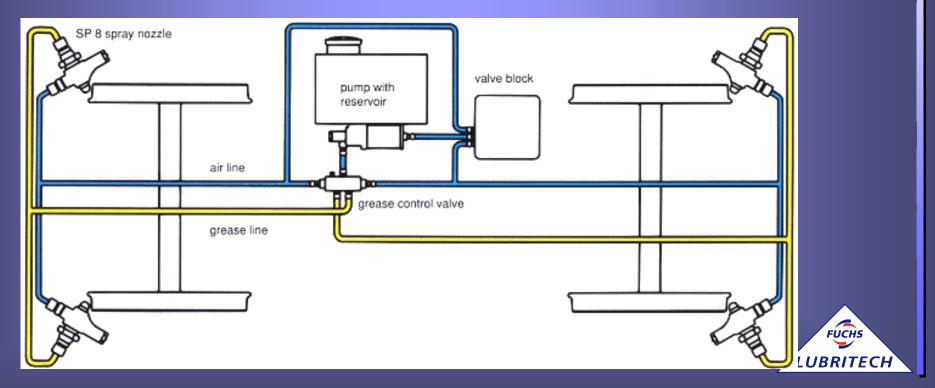




# Systems SP 8 – System with Pump

- **Expansive system for universal use**
- Pneumatic pump for long line lengths
- Sturdy reliable design
- **Requires little maintenance**

| Technical Data        |                                   |
|-----------------------|-----------------------------------|
| Distribution          | up to 20m main line (pump/nozzle) |
| Operating pressure    | 100 bars                          |
| Reservoir size        | customer                          |
| Temperature range     | -25 to 80° C                      |
| Lubricant             | fluid grease NLGI grade 000 or 00 |
| Metering              | 0.15; 0.05; 0.03 ccm/spray        |
| Application frequence | zy max 2min                       |



# **Tangent track lubrication**

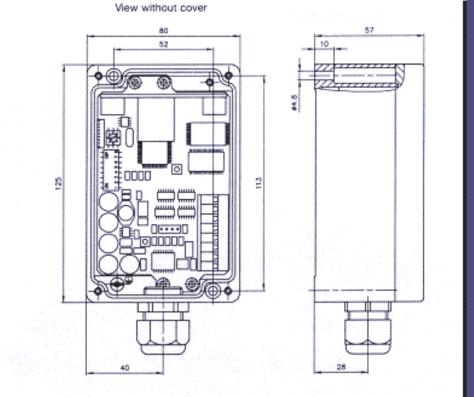
Optimal supply throughout entire rail road networkAdjustable for 100, 200 and 300 meter lubrication interval

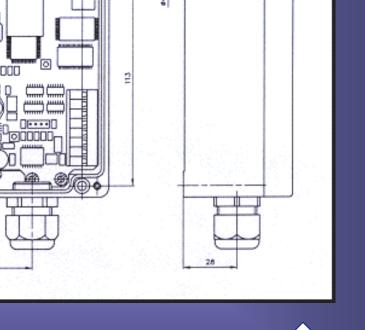
# **Curve sensitive lubrication**

Lubricates only in critical areas of curves
Self contained system – no locomotive interface required
Electronic curve sensor – no moving part
Minimal lubricant consumption

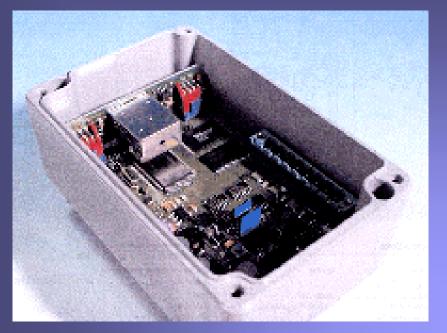


# **CS200** Curve Sensor **Control Unit**









#### Modes of Operation

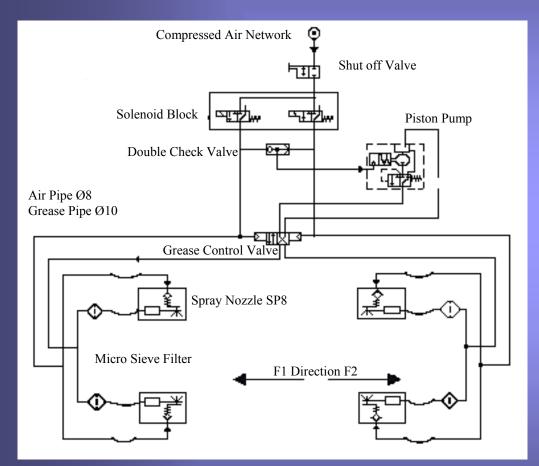
Curve lubrication Time controlled lubrication Distance controlled lubrication Curve / time controlled lubrication Randomly controlled lubrication after power up Curve sensor for external control unit

•Powerful valve outputs •Adjustable sensitivity depending on practical needs

#### **Practical examples**

Stand-alone wheel flange control unit Curve sensor to supplement wheel flange control units Curve sensor for lubrication control by railway SPC

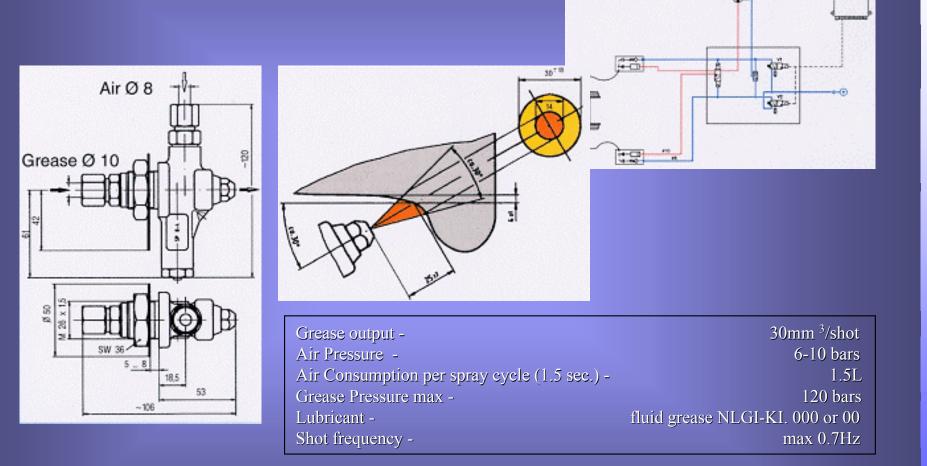
# **VOGEL – Wheel Flange Lubrication Systems SP 8**



- High Pressure system (max120 bar) for long pipes between pump and nozzle
- Pneum. Grease pump (5/25kg ratio 1:13
- Grease control valve for forward and reverse or left and right hand lubrication
- Lubrication module (pump, SF10 solenoids)
- Distance /time depending lubrication with IG 666(24, 110VDC
- For oils and semisolid greases NLGI 000...1



# Spray Nozzle SP 8-4/S3: Dimensions and Adjustments





# <u>There Are Basically Two Types of Systems</u>

Lubricant is dispensed into a single feed line with the air, the lubricant is atomized and carried to the spray nozzle then out to the wheel flange

# <u>Problem</u>

Changes in temperature will affect the apparent viscosity of the grease which will change the amount of lubricant dispensed. A further complication is with long and difficult routed tube runs the amount of lubricant dispensed is compromised.

# <u>The Vogel Way is:-</u>

• A positively dispensed shot of grease with an independent air supply mixing the lubricant at the spray nozzle, ensuring precise lubrication at all times unaffected by temperature change or installation routing



# Wheel flange lubrication, calculation example:



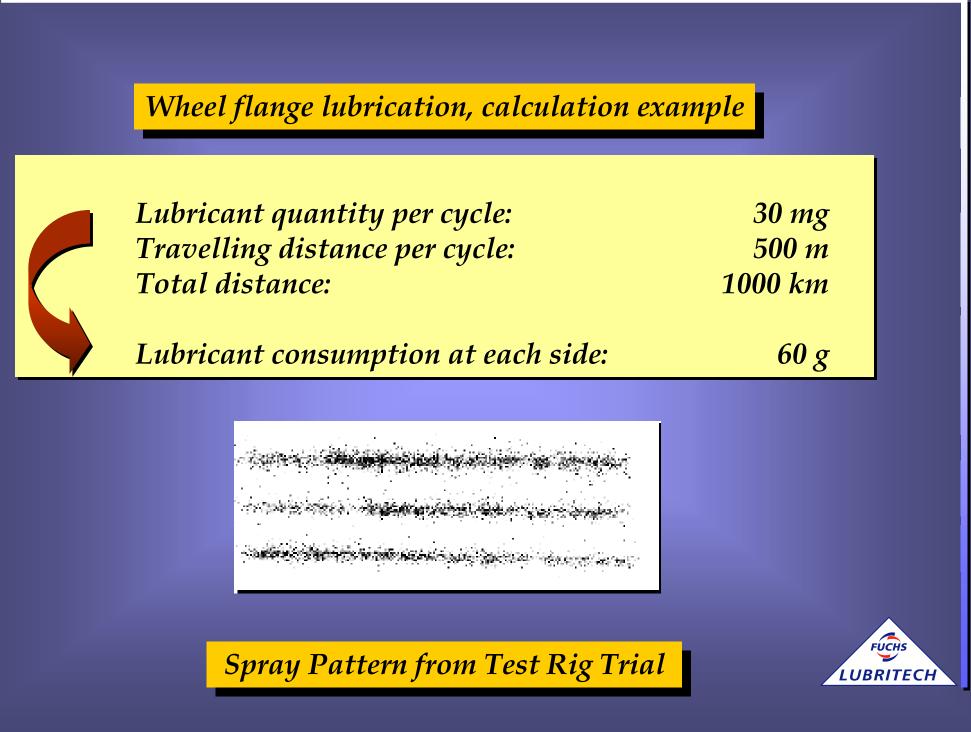
| Width of the spray jet on the wheel flange: | <b>20</b> mm |
|---|--------------|
| Wheel diameter:                             | 900 mm       |
| Lubricant quantity per spray cycle:         | 30 mg        |

*Thickness of lube film on the first wheel set: ca. 0.5 µm* 

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Spray Pattern from Test Rig Trial





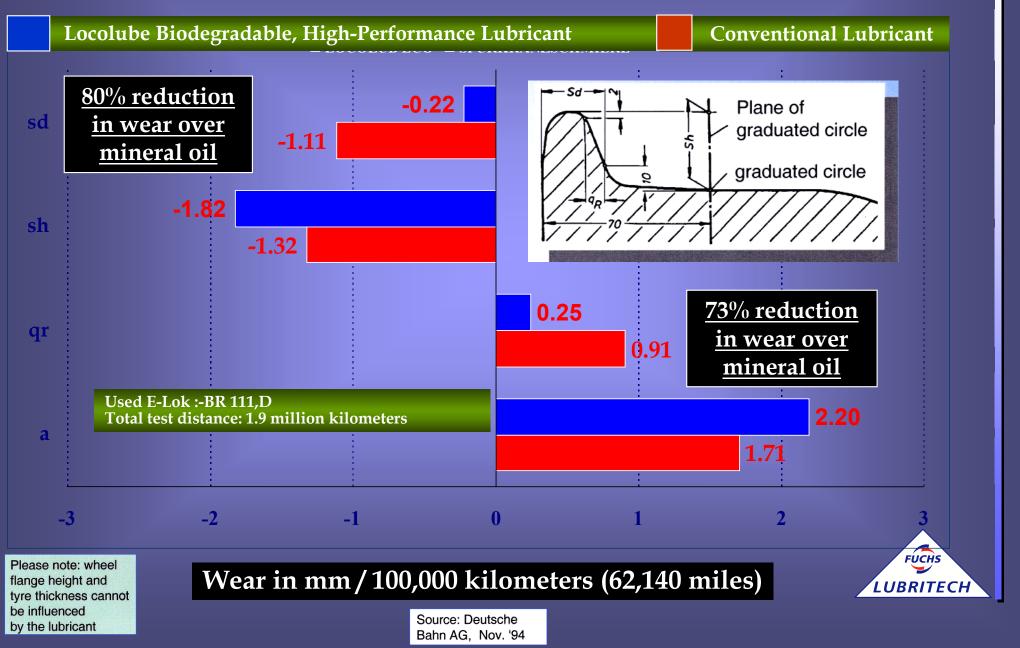
# Conrails Eyeball Chart

# Visual Determination of the Coefficient of Friction

| OBSERVED CONDITION  | APPROXIMATE µ |
|---|---------------|
| Chewed up rough gauge face.   | > 0.45        |
| Smooth gauge face with a shiny un-lubricated surface.               | 0.35 to 0.45  |
| Smooth gauge face with lubricant covering 10 to 40% of the surface. | 0.30 to 0.35  |
| Smooth gauge face with lubricant covering 40 to 60 % of the surface |               |
| metal still visible through the lubricating film                    | 0.25 to 0.30  |
| Smooth gauge face with lubricant covering 60 to 90 % of the surface | 0.20 to 0.25  |
| Gauge face 100% covered with a thin film of lubricant, polished     |               |
| surface of the rail may be seen through the lubricant               | 0.15 to 0.20  |
| Gauge face 100% covered by a black lubricating film of graphite     | < 0.15        |



# Field Test Results

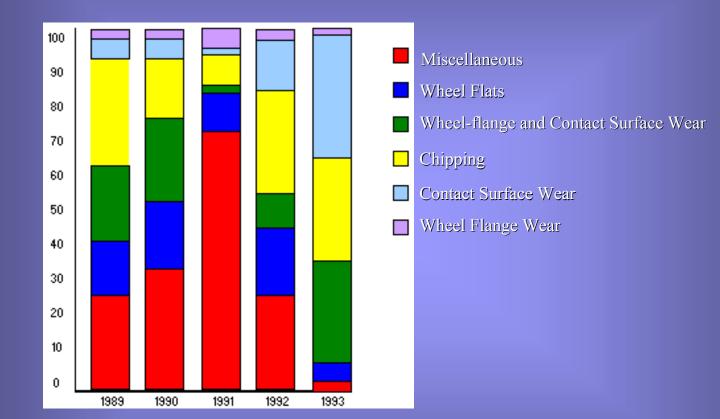


# Reduction of wheel flange wear

| without WFL                          | <b>Published Results</b> | Operator | Without WFL | With W F L    |
|--------------------------------------|--------------------------|----------|-------------|---------------|
|                                      | Re-profiling period      | D.B. AG  | 480,000 km  | 960,000 km    |
|                                      | Wheel Flange repair      | D.B. AG  |             | 50% Reduction |
|                                      |                          | OBB      |             | 50% Reduction |
| with WFL                             |                          | Conrail  |             | 70% Reduction |
|                                      | Re-profiling costs       | D.B. AG  |             | 50% Reduction |
|                                      |                          | OBB      |             | 60% Reduction |
| metal removal with change of contour |                          | Conrail  |             | 20% Reduction |



# **Cause – Related Treatment of Wheel Sets**





|                        | Conrail | CSX  | AAR | Key Factors   |
|------------------------|---------|------|-----|---|
| Fuel/Energy savings    | 25 35%  | 810% | 35% | • Quality of bogies and tracks                                  |
| Maintence cost cuts    |         |      |     | Reliable lubrication systems                                    |
| for wheel sets         | DB AG   | BN   | ÖBB | Vehicles with WFL   |
|                        | 50%     | 80%  | 30% |   |
| Lubricant savings ref: |         |      |     |   |
| to wayside lubricator  | 20%     | 50%  | 50% | Potential Annual Savings From                                   |
|                        |         |      |     | Potential Annual Savings From<br>Rail Lubrication (\$ Millions) |
| Labour costs cuts ref: |         |      |     |   |
| To wayside lubricator  | 3040%   | -    | 50% |   |

| Potential Annual Savings From Rail Lubrication (\$ Millions) |                |             |  |  |
|--|----------------|-------------|--|--|
|  | All Main Track | Curves only |  |  |
| Energy   | 300-400        | 90-150      |  |  |
| Rail   | 245            | 245         |  |  |
| <u>Wheels</u>  | -70            | -15         |  |  |
| Net savings  | 530-670        | 370-430     |  |  |



# **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Fiat Ferroviaria Number of Vehicles – 35 Number of SP8 Systems – 70 **High Speed Train ETR 500 For FS** 





# **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Breda Number of Vehicles – 145 Number of SP8 Systems – 145 **E-Loco E633 For FS** 





# **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Siemens SGP Number of Vehicles – 50 Number of SP8 Systems – 50 **Shunting Loco Type 1063 For ÖBB** 





# **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Breda Number of Vehicles – 410 Number of SP8 Systems – 410 **E-Loco E656 For FS** 





### **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Ansaldo Number of Vehicles – 230 Number of SP8 Systems – 230 **E-Loco E402 for FS** 





# **VOGEL – Wheel Flange Lubrication Systems SP 8**



Manufacturer – Adtranz Number of Vehicles – 120 Number of SP8 Systems – 120 **Diesel Loco DG1200 for Private Railways** 





# **Future Requirements**

- Surther development of existing products and systems
- Search for higher performance raw materials
- Solution Take account of new trends in environmental legislation
- Be aware of development of new materials for wheels and rails
- Be aware of new designs covering carriages and bogies
- Be aware of future requirements for higher wheel loads



- Track resistance drops by some 30 to 35%, and motive-power reserves are increased. Traction and braking performance are not affected.
- Fuel savings up to 15% create fast R.O.I. in less than six months, improvement in shareholder value.
- Wear reduction by 30 to 80%. Time between wheel truing and rail life is extended.
- Greater safety due to the decreased possibility of derailment, it is harder for the wheel flange to climb the track.





